



The Open University of Sri Lanka
Faculty of Engineering Technology
Department of Civil Engineering

080

Study Programme	: Bachelor of Technology Honours in Engineering
Name of the Examination	: Final Examination
Course Code and Title	: CVX5533 Structural Analysis
Academic Year	: 2020/21
Date	: 27 th January 2022
Time	: 0930-1230hrs
Duration	: 3 hours

General Instructions

1. Read all instructions carefully before answering the questions.
2. This question paper consists of **Eight (8)** questions in **Five (5)** pages.
3. Answer any **Five (5)** questions only. All questions carry equal marks.
5. Answer for each question should commence from a new page.
6. This is Closed Book Test (CBT).
7. Answers should be in clear hand writing.
8. Do not use Red colour pen.

QUESTION 1

- (i) Briefly describe “**how stress is not a vector**” with neat sketches (3 Marks)
- (ii) Strain tensor for a homogenous, isotropic material is given below.

$$\sigma_{ij} = \begin{bmatrix} -100 & 0 & -80 \\ 0 & 20 & 0 \\ -80 & 0 & 20 \end{bmatrix}$$

- (a) Write six independent stress components (3 Marks)
- (b) Determine three stress invariants (4 Marks)
- (c) Determine three principal stresses (5 Marks)
- (d) Determine three principal strains if the elastic modulus and Poisson’s ratio are 200 GPa and 0.3, respectively. (5 Marks)

QUESTION 2

- (i) Write three stress components according to the Airy’s stress function with usual notation. (3 Marks)
- (ii) A mathematical function is given below.

$$\phi = Ay^3 + Bxy + Cxy^3$$

where A, B and C are constants.

- (a) Show that ϕ is an admissible stress function for a cantilever beam with a concentrated load of **P** at the end. The beam has cross section dimensions of width “**b**” and depth “**d**”, respectively. The beam length is “**L**”. (4 Marks)
- (b) State the boundary conditions. (6 Marks)
- (c) Determine three stress components. (4 Marks)
- (d) Briefly explain the limitations of the above stress function. (3 Marks)

QUESTION 3

- (i) Explain three characteristics of statically indeterminate structures (4 Marks)
- (ii) A continuous beam (ABCD) is shown in Figure Q3. Flexural rigidities of members AB and CD are equal to $2EI$ and member BC is $4EI$. Uniformly distributed loads (W), ($2W$) are acting on members, AB and BC, respectively. There is a concentrated load ($2Wl$) in member CD.
- a) Determine the degree of the statical indeterminacy of the beam. (3 Marks)

- b) Draw a released structure. (3 Marks)
- c) Determine the flexibility matrix for the drawn released structure. (4 Marks)
- d) Determine bending moments at B and C. (6 Marks)

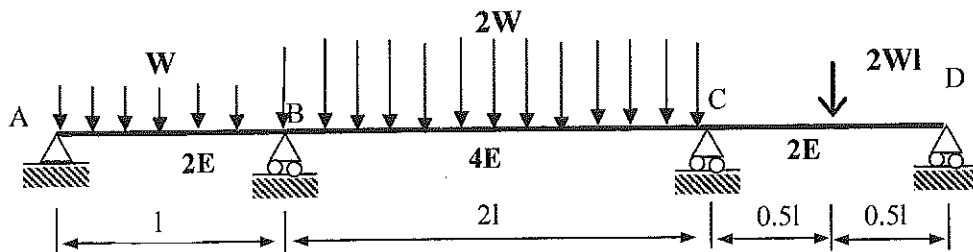


Figure Q3

QUESTION 4

- (i) A portal frame structure shown in Figure Q4. All members have same flexural rigidity (EI). You can neglect the axial deformation.

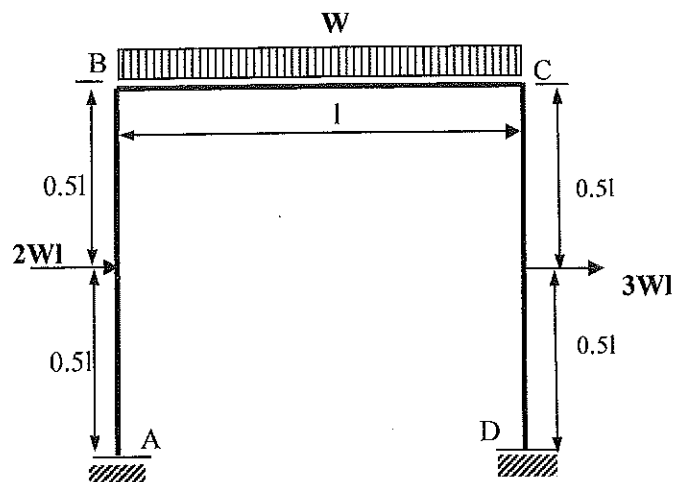


Figure Q4

- (i) Find Kinematic indeterminacy of the structure (2 Marks)
- (ii) Draw the structure with independent nodal displacements (2 Marks)
- (iii) Determine the stiffness matrix of the structures. (4 Marks)
- (iv) Find the free nodal displacements at B using the displacement method. (6 Marks)
- (v) Using above results, determine the bending moment at B. (6 Marks)

QUESTION 5

- (i) List three methods of experimental stress analysis methods. (2 Marks)
- (ii) Briefly explain the difference between **null method** and **out of balance method** in experimental stress measurements using electrical resistance strain gauges. (3 marks)
- (iii) The stress state of a certain steel components was determined using a strain rosette as shown in Figure Q5. Due to the loadings, strain gauges gave strain values as $\epsilon_a = 30 \mu\epsilon$, $\epsilon_b = 50 \mu\epsilon$, $\epsilon_c = 90 \mu\epsilon$.

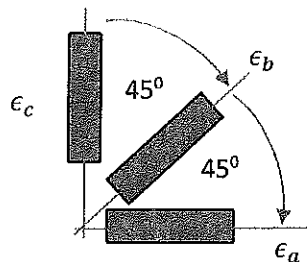


Figure Q5

- (a) Draw the Mohr's circle for strains (4 marks)
- (b) Determine the in-plane principal strains. (5 marks)
- (c) Determine the principal stress, if the bracket material is steel ($E = 200\text{GPa}$, $\nu = 0.3$) (6 marks)

QUESTION 6

- (i) Briefly describe three conditions used in plastic analysis of structures. (2 Marks)
- (ii) A two-bay frame structure is shown in Figure Q6. Dimensions and plastic moments of the columns and beam are given in the figure.
- (a) Draw possible locations of plastic hinge formations. (2 Marks)
- (b) Draw elementary failure mechanisms. (2 Marks)
- (c) Determine load factors for each elementary failure mechanism. (6 Marks)
- (d) Determine the most probable failure mechanism by combining elementary failure mechanisms. (6 Marks)
- (e) Explain how you can ensure the unique solution. (2 Marks)

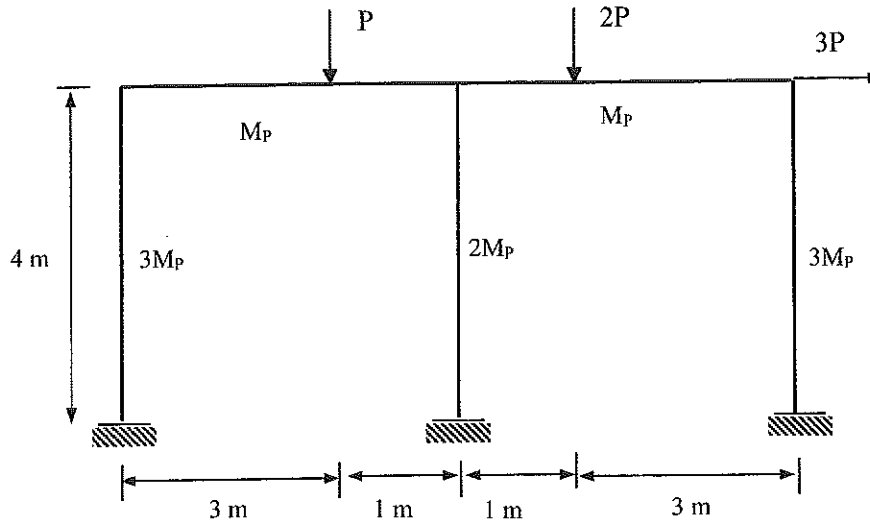


Figure Q6

QUESTION 7

Write short notes on following topics

- (i) The difference between static and dynamic loadings and how their applications in engineering structures. (4 Marks)
- (ii) The importance of applications of finite element methods in structural analysis (4 Marks)
- (iii) The difference between elastic and plastic neutral axes in plastic analysis of beams (4 Marks)
- (iv) Application of membrane theory in structural analysis (4 Marks)
- (v) Difference between Kinematic indeterminacy and Statical Indeterminacy (4 Marks)

QUESTION 8

The Governing equation for plate bending is given by the following expressions

$$\frac{\partial^4 \omega}{\partial x^4} + \frac{2\partial^4 \omega}{\partial x^2 \partial y^2} + \frac{\partial^4 \omega}{\partial y^4} = \frac{q}{D}$$

All the terms have their normal meanings.

- (i) Write three main assumptions used in deriving the plate bending governing equation. (4 Marks)
- (ii) A simply supported rectangular plate (Figure Q8) is subjected to a sinusoidal load

$$q = q_0 \sin \frac{\pi x}{a} \sin \frac{\pi y}{b}.$$

If the deflection of the plate can be expressed as $\omega = C \sin \frac{\pi x}{a} \sin \frac{\pi y}{b}$

Show that ω is given by the following expression,

$$\omega = \frac{q_o}{D\pi^4 \left(\frac{1}{a^2} + \frac{1}{b^2} \right)^2} \sin \frac{\pi x}{a} \sin \frac{\pi y}{b} \quad (8 \text{ Marks})$$

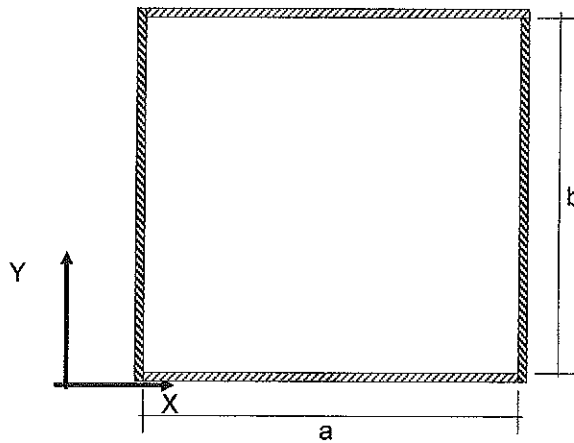


Figure Q8

- (iii) The first successful solution for the governing equation for plate bending was proposed by Navier in 1820.

$$\text{If } q = q_{(x,y)} = \sum_{m=1}^{\infty} \sum_{n=1}^{\infty} a_{mn} \sin \frac{m\pi x}{a} \sin \frac{n\pi y}{b}$$

Show that any particular coefficient (a_{mn}) is given by the following expression

$$a_{mn} = \frac{4}{ab} \int_0^a \int_0^b q_{(x,y)} \sin\left(\frac{m\pi x}{a}\right) \sin\left(\frac{n\pi y}{b}\right) dx dy \quad (5 \text{ Marks})$$

You can assume that,



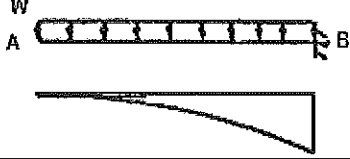


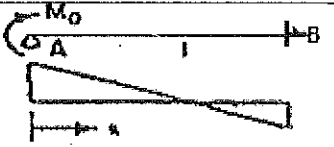
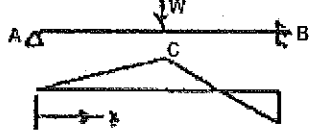
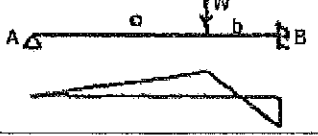
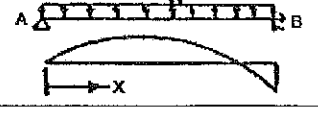
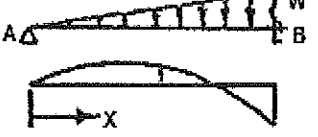

$$\int_0^b \sin\left(\frac{n\pi y}{b}\right) \sin\left(\frac{n'\pi y}{b}\right) dy = b/2 \text{ when } n=n' \text{ else } \int_0^b \sin\left(\frac{n\pi y}{b}\right) \sin\left(\frac{n'\pi y}{b}\right) dy = 0$$

Hence show that for uniformly distributed load

$$a_{mn} = \frac{4q}{\pi^2 mn} \text{ when } m=1, 3, 5 \text{ and } n=1, 3, 5$$

and for even number of m and n $a_{mn} = 0$ (3 Marks)

Formulas for Beams				
Structure	Shear \uparrow	Moment \cup	Stop \times	Deflection \downarrow
Simply supported Beam				
	$S_A = -\frac{M_o}{L}$	M_o	$\theta_A = \frac{M_o L}{3EI}$ $\theta_B = \frac{M_o L}{6EI}$	$Y_{max} = 0.062 \frac{M_o L^2}{3EI}$ $Y_c = \frac{WL^3}{48EI}$
	$S_A = \frac{W}{2}$	$M_o = \frac{WL}{4}$	$\theta_A = -\theta_B = \frac{WL^2}{16EI}$	$Y_c = \frac{WL^3}{48EI}$
	$S_A = \frac{Wb}{L}$ $S_B = \frac{Wn}{L}$	$M_o = \frac{Wab}{L}$	$\theta_A = \frac{Wab}{6EI L} (L + b)$ $\theta_B = -\frac{Wab}{6EI L} (L + a)$	$Y_o = \frac{Wa^2b^2}{3EI L}$
	$S_A = \frac{WL}{2}$	$M_c = \frac{WL^2}{8}$	$\theta_A = -\theta_B = \frac{WL^3}{24EI}$	$Y_c = \frac{5WL^4}{384EI}$
	$S_A = \frac{WL}{6}$ $S_B = \frac{WL}{3}$	$M_{max} = 0.064WL^2$ at $x = 0.577L$	$\theta_A = \frac{7WL^3}{360EI}$ $\theta_B = -\frac{8WL^3}{360EI}$	$Y_{max} = 0.00652 \frac{WL^4}{EI}$ at $x = 0.519L$
	$S_A = \frac{WL}{4}$	$M_c = \frac{WL^2}{12}$	$\theta_A = -\theta_B = \frac{5WL^3}{192EI}$	$Y_c = \frac{WL^4}{120EI}$
Fixed Beams				
	$S_A = \frac{W}{2}$	$M_c = \frac{WL}{8}$	$\theta_A = \theta_B = 0$	$Y_c = \frac{WL^3}{192EI}$
	$S_A = \frac{Wb^2}{L^3} (3a + b)$ $S_B = \frac{Wa^2}{L^3} (3b + a)$	$M_A = -\frac{Wab^2}{L^2}$ $M_B = -\frac{Wba^2}{L^2}$	$\theta_A = \theta_B = 0$	$Y_o = \frac{Wa^3b^3}{3EIL^3}$
	$S_A = \frac{WL}{2}$	$M_A = M_B = -\frac{WL^2}{12}$	$\theta_A = \theta_B = 0$	$Y_c = \frac{WL^4}{384EI}$
	$S_A = \frac{3WL}{20}$ $S_B = \frac{7WL}{20}$	$M_A = -\frac{WL^2}{30}$	$\theta_A = \theta_B = 0$	$Y_{max} = 0.00131 \frac{WL^4}{EI}$ at $x = 0.525L$
	$S_A = \frac{WL}{4}$	$M_A = M_B = -\frac{SWL^2}{96}$	$\theta_A = \theta_B = 0$	$Y_c = \frac{0.7WL^4}{384EI}$

Structure	Shear ↑↓	Moment ∪∩	Stop ✕	Deflection ↓
Cantilever Beam				
	0	M_o	$\theta_A = \frac{M_o L}{EI}$	$Y_A = \frac{M_o L^2}{2EI}$
	W	$M_B = -WL$	$\theta_A = -\frac{WL^2}{2EI}$	$Y_A = \frac{WL^3}{3EI}$
	$S_B = -WL$	$M_B = -\frac{WL^2}{2}$	$\theta_A = \frac{WL^3}{6EI}$	$Y_A = \frac{WL^4}{8EI}$
	$S_B = -\frac{WL}{2}$	$M_B = -\frac{WL^2}{6}$	$\theta_A = -\frac{WL^3}{24EI}$	$Y_A = \frac{WL^4}{8EI}$
	$S_B = -\frac{WL}{2}$	$M_B = -\frac{WL^2}{2}$	$\theta_A = -\frac{WL^3}{8EI}$	$Y_A = \frac{11WL^4}{120EI}$
Propped Cantilever				
	$S_A = \frac{3M_o}{2L}$	$M_B = \frac{M_o}{2}$	$\theta_A = -\frac{M_o L}{4EI}$	$Y_{max} = \frac{W_o L^2}{27EI}$ at $x = \frac{L}{3}$
	$S_A = -\frac{5W}{16}$	$M_B = -\frac{3WL}{16}$ $M_C = -\frac{5WL}{32}$	$\theta_A = -\frac{WL^2}{32EI}$	$Y_{max} = 0.00932 \frac{WL^3}{EI}$ at $x = 0.447L$
	$S_A = \frac{Wb^2}{2L^3}(a+2L)$ $S_B = \frac{Wa}{2L^3}(3L^2-a^2)$	$M_B = -\frac{Wab}{L^2}\left(a+\frac{b}{2}\right)$	$\theta_A = -\frac{Wab^3}{4EIL}$	$Y_o = \frac{Wa^2b^3}{12EIL^3}(3L+a)$
	$S_A = \frac{3WL}{8}$	$M_B = -\frac{WL^2}{8}$	$\theta_A = -\frac{WL^3}{48EI}$	$Y_{max} = 0.0054 \frac{WL^4}{EI}$ at $x = 0.422L$
	$S_A = \frac{WL}{10}$	$M_{max} = 0.03WL^2$ at $x = 0.447L$ $M_B = -\frac{WL^2}{15}$	$\theta_A = -\frac{WL^3}{120EI}$	$Y_{max} = 0.00239 \frac{WL^4}{EI}$ at $x = 0.447L$
	$S_A = \frac{11WL}{40}$	$M_{max} = 0.0423WL^2$ at $x = 0.329L$ $M_B = -\frac{7WL^2}{120}$	$\theta_A = -\frac{WL^3}{80EI}$	$Y_{max} = 0.00305 \frac{WL^4}{EI}$ at $x = 0.402L$