

The Open University of Sri Lanka

Faculty of Engineering Technology



Study Programme	: Bachelor of Technology Honours in Engineering
Name of the Examination	: Final Examination
Course Code and Title	: MEX 6278 – FLUID MECHANICS
Academic Year	: 2016/17
Date	: 09 th , December 2017
Time	: 9.30am -12.30pm
Duration	: 3 hours

General Instructions

1. Read all instructions carefully before answering the questions.
2. This question paper consists of 8 questions. All questions carry equal marks.
3. Answer **any 5** questions only.
4. Take acceleration due to gravity and the density of water as **9.81 N/kg** and **1000 kg/m³** respectively where necessary.

Q1.

- a) By considering a differential fluid element, show that the vector form of the continuity equation for a compressible, unsteady fluid flow with usual notation is given by,

$$\frac{d\rho}{dt} + \nabla(\rho u) = 0$$

6-marks

Where $\nabla = \left(\frac{\partial}{\partial x} i + \frac{\partial}{\partial y} j + \frac{\partial}{\partial z} k \right)$

- b) Obtain the expressions for the continuity equations of following fluid flows.
- i. Steady compressible.
 - ii. Steady incompressible.
- c) The velocity components in x and y direction of an incompressible, steady-flow field are $u = x^2 + y^2 + z^2$ and $v = xy + yz + z$ respectively. Determine the velocity component in the z direction (w), required to satisfy the law of conservation of mass.

4-marks

10-marks

Q2.

- (a). By considering velocity variation that causes rotation and angular deformation in a two dimensional differential fluid element shown in Figure Q2, Show that the angular velocity of the fluid flow is given by,

$$\omega = \frac{1}{2} \nabla \times V$$

10-marks

where $\omega = \omega_x \hat{i} + \omega_y \hat{j} + \omega_z \hat{k}$ and $V = v_x \hat{i} + v_y \hat{j} + v_z \hat{k}$

10-marks

- (b). A velocity field in a flow is given by $V = 2xy\mathbf{i} + 4tz^2\mathbf{j} - yz\mathbf{k}$ ms^{-1} . Find the acceleration, the angular velocity about z-axis, and the vorticity vector at the point (2m, -1m, 1m) at $t=2$ sec.

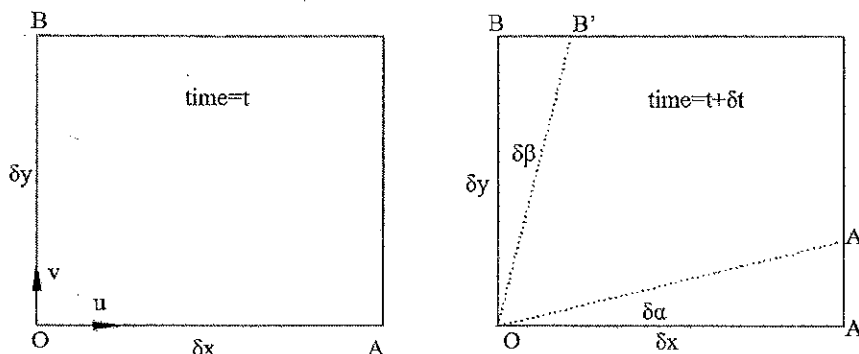


Figure Q2.

Q3.

The differential pressure (dP) of a forced vortex motion in a cylindrical container, with usual notation, is given by,

$$dP = \rho r \omega^2 dr - \rho g dz$$

An open cylindrical container **0.5m** in diameter and **0.8m** in height, filled with oil up to **0.5m** and is rotating about its vertical axis. The specific gravity of liquid is **0.88**.

6-marks

- (a). Show that the water surface takes the form of a paraboloid.
 (b). Calculate the speed at which the liquid will start to spill over.
 (c). Calculate the speed at which the point of the bottom centre will just expose?

8-marks

6-marks

Q4.

(a). State the Buckingham's π theorem.

2-marks

(b). The pressure drop (Δp) across a gate valve depends on the gate opening h , overall depth d , velocity V , density of fluid ρ , viscosity of the fluid μ .

Using Buckingham's π theorem, show that the variables can be expressed as,

$$\Delta p = \rho V^2 f\left(\frac{h}{d}, \frac{\rho V d}{\mu}\right)$$

8-marks

(c). What is dynamic similarity?

2-marks

(d). A prototype gate valve which will control the flow in a pipe system that conveys paraffin is to be studied in a model. A 1:5 scale model is built to determine the pressure drop across the valve with water as the working fluid.

4-marks

I. For a certain valve opening, when the velocity of paraffin in the prototype is 3.0 ms^{-1} . What should be the velocity of water in the model for dynamic similarity?

4-marks

II. What is the pressure drop in the prototype if it is 60 kPa in the model?

(The density and the viscosity of paraffin are 800 kgm^{-3} and $0.002 \text{ kgm}^{-1}\text{s}^{-1}$ respectively. The kinematic viscosity of water is $1.0 \times 10^{-6} \text{ m}^2 \text{ s}^{-1}$).

Q5.

3-marks

(a). Draw a schematic diagram of boundary layer for a laminar flow over a smooth flat plate clearly showing the boundary layer thickness (δ), velocity profile $u(y)$ and the free stream velocity u_∞ .

3-marks

(b). Explain the significance of the boundary layer displacement thickness (δ^*) and the momentum thickness (Θ), and express them as integrals of the boundary layer velocity profiles for a smooth flat plate.

7-marks

(c). A laminar boundary layer is given by the following velocity profile,

$$\frac{u}{u_\infty} = \alpha \left(\frac{y}{\delta}\right) + \beta \left(\frac{y}{\delta}\right)^3$$

Where α and β are arbitrary constants

i. Find α and β by using the boundary conditions

7-marks

ii. Show that $(\delta^*/\delta) = 3/8$

Q6.

Consider the laminar flow of a fluid layer falling down a plane inclined at an angle θ to the horizontal as shown in Figure Q6.

Starting from Navier-Stokes equations

7-marks

- (a). Show that for steady flow the momentum equation in x-direction can be given expressed as ,

$$\rho g \sin \theta + \mu \frac{\partial^2 u}{\partial y^2} = 0$$

- (b). If h is the thickness of the layer in the fully developed stage, show that the velocity distribution is given by,

$$u(y) = \frac{-\rho g \sin \theta}{2\mu} (y^2 - h^2)$$

7-marks

- (c). Show that the volume flow rate per unit width is given by,

$$Q = \left(\frac{\rho g h^3}{3\mu} \right) \sin \theta$$

6-marks

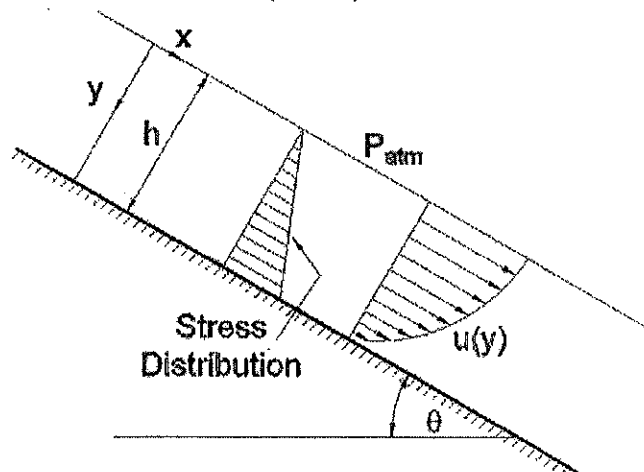


Figure Q6

Q7.

- (a). State the applications of potential flow theory.

4-marks

- (b). Define stream function and velocity potential and show that each of them satisfy Laplace's equation.

4-marks

- (c). A flow field is characterized by the stream function $\psi = Axy$ where $A = 2\text{s}^{-1}$ and the coordinates are measured in meters. Verify that the flow is irrotational and determine the velocity potential ϕ and sketch the streamlines and potential lines.

12-marks

Q8

(a). Describe the following terms,

I. Doublet

II. Rankine Oval

III. Singularity

6-marks

(b). A source of strength m is placed at the origin of a uniform flow field (U) as shown in Figure Q8.

7-marks

Show that the resultant stream function and the velocity potential respectively are given by,

$$\psi = Ur \sin \theta + \frac{m}{2\pi} \theta \quad \text{and} \quad \phi = Ur \cos \theta + \frac{m}{2\pi} \ln(r)$$

(c). A stagnation point is created at $(-b, 0)$, show that the equation of the stream line passing through this stagnation point is given by,

$$r = \frac{b(\pi - \theta)}{\sin \theta}$$

7-marks

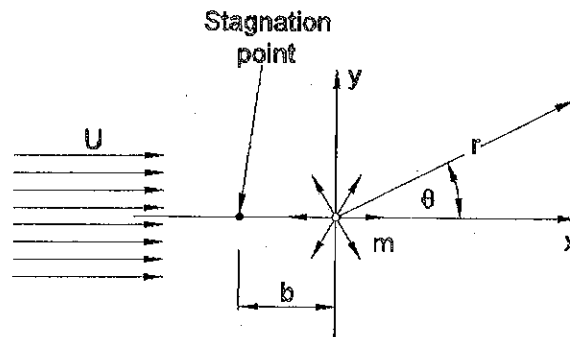


Figure Q8

Navier-Stokes equations for incompressible flow:-

$$\rho \left(\frac{\partial u}{\partial t} + u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} + w \frac{\partial u}{\partial z} \right) = \rho g_x - \frac{\partial p}{\partial x} + \mu \left(\frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2} + \frac{\partial^2 u}{\partial z^2} \right)$$

$$\rho \left(\frac{\partial v}{\partial t} + u \frac{\partial v}{\partial x} + v \frac{\partial v}{\partial y} + w \frac{\partial v}{\partial z} \right) = \rho g_y - \frac{\partial p}{\partial y} + \mu \left(\frac{\partial^2 v}{\partial x^2} + \frac{\partial^2 v}{\partial y^2} + \frac{\partial^2 v}{\partial z^2} \right)$$

$$\rho \left(\frac{\partial w}{\partial t} + u \frac{\partial w}{\partial x} + v \frac{\partial w}{\partial y} + w \frac{\partial w}{\partial z} \right) = \rho g_z - \frac{\partial p}{\partial z} + \mu \left(\frac{\partial^2 w}{\partial x^2} + \frac{\partial^2 w}{\partial y^2} + \frac{\partial^2 w}{\partial z^2} \right)$$

Velocity potential and stream function in cylindrical coordinates:

$$v_r = \frac{\partial \phi}{\partial r} = \frac{1}{r} \frac{\partial \psi}{\partial \theta}$$

$$v_\theta = \frac{1}{r} \frac{\partial \phi}{\partial \theta} = -\frac{\partial \psi}{\partial r}$$

End of Paper