

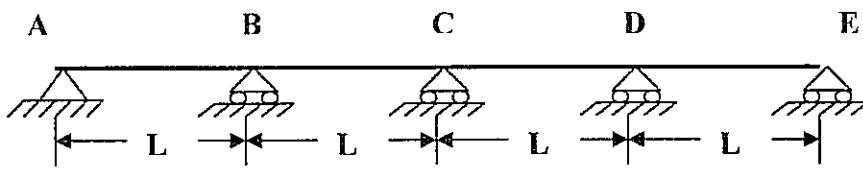


Date: 2006-04-05 (Wednesday)
hrs.

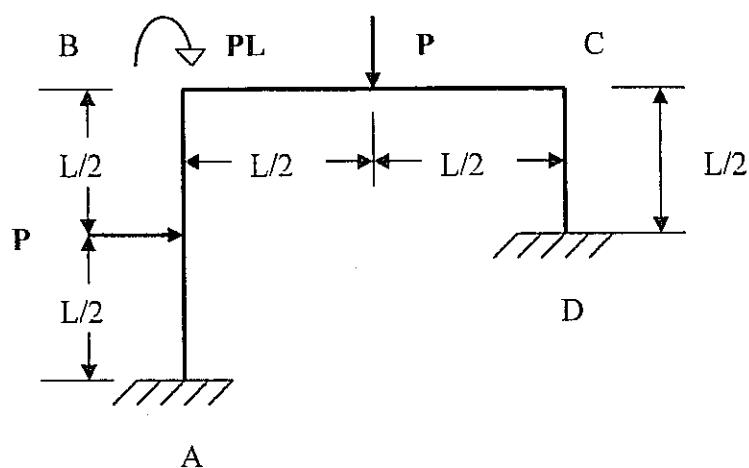
Time: 09.30 – 12.30

The Paper consists of Eight (8) questions. Answer Five (5) questions

1. Analyse the continuous beam shown below, using flexibility method, for
 (a) a uniformly distributed load of intensity q on all spans
 (b) a unit downward movement of support B
 and draw the bending moment diagram.
 The beam has a constant flexural rigidity EI .
 (You may use Table-1 to determine flexibility coefficients)



2. The plane frame shown in consists of rigidly connected members of constant flexural rigidity EI . Analyse the frame using displacement method.
 Neglect axial deformation.
 (You may use Table-1 to determine stiffness coefficients)



3. The components of stress at a point in a body are $\sigma_x = 0$; $\sigma_y = 300$; $\sigma_z = 100$; $\tau_{xy} = \tau_{xz} = 0$; $\tau_{yz} = 100\sqrt{3}$ N/m².

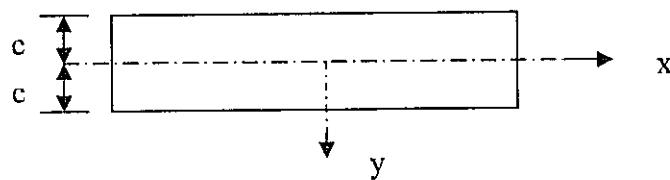
Determine ;

- (a) the stress invariants
- (b) the principal stresses
- (c) the direction cosines of the principal planes

4. Show that $\phi = \frac{q}{8c^3} \left[x^2(y^3 - 3c^2y + 2c^3) - \frac{1}{5}y^3(y^2 - 2c^2) \right]$ is a stress function.

Determine the stress components σ_x , σ_y and σ_{xy} .

Find what problem it solves when applied to the region included in $y = \pm c$, $x = 0$, on the side x positive.



5. A conical tank with semi-cone angle α and height h is filled with a liquid of specific weight γ . Show that the stress resultants are given by

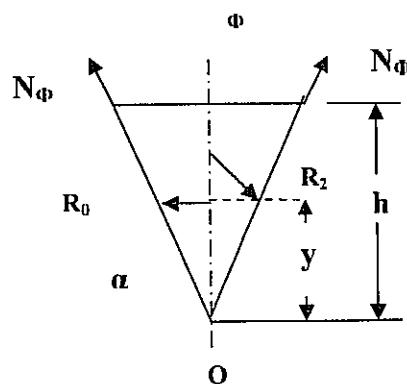
$$N_\theta = \frac{\gamma(h-y)y \tan \alpha}{\cos \alpha}$$

$$N_\phi = \frac{\gamma y(h - \frac{2}{3}y) y \tan \alpha}{2 \cos \alpha}$$

Also show that the maximum N_θ is when $y = d/2$ and maximum N_ϕ is at $y = 3d/4$. Determine N_θ^{\max} and N_ϕ^{\max} .

Membrane solution for shells of revolution is given by;

$$\frac{N_\theta}{R_2} + \frac{N_\phi}{R_1} = -p_r$$

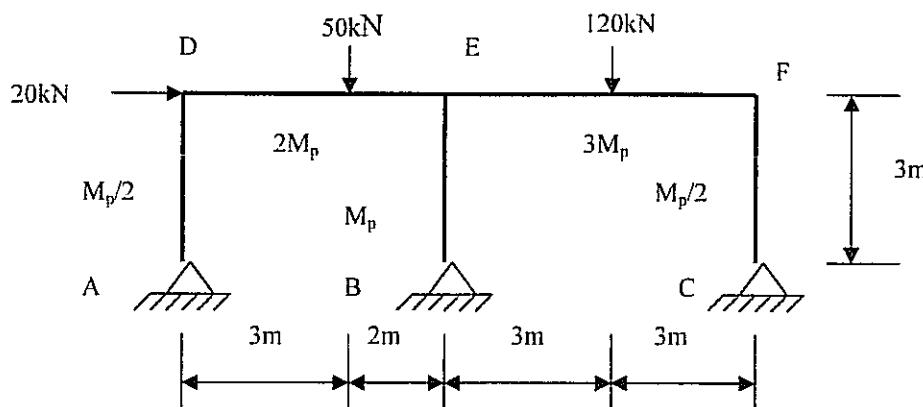


6. A circular plate of radius a is carrying a concentrated load P at the centre. Assuming that the edge of the plate is built-in, show that the deflection of the plate is given by;

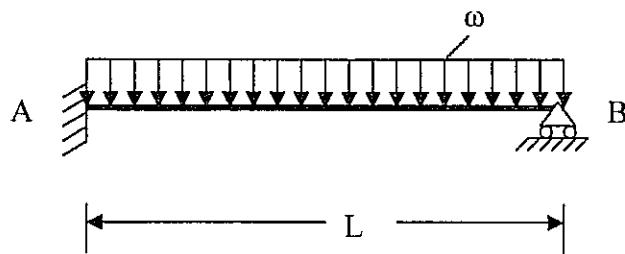
$$\omega = \frac{Pr^2}{8\pi D} \log \frac{r}{a} + \frac{P}{16\pi D} (a^2 - r^2)$$

Hence determine the maximum deflection of the plate.
Also find the moment M_r at built-in edge.

7. Analyse the rigid frame shown below using plastic method of analysis and determine the Plastic Moment M_p , for a load factor of 1.8.



8. (a) Determine the collapse load for the fixed beam shown in the figure below, if the Plastic Moment Capacity of the beam is M_p .



- (b) Determine the ultimate load for the propped cantilever shown in the figure below, if the Plastic Moment Capacity of the beam is M_p .

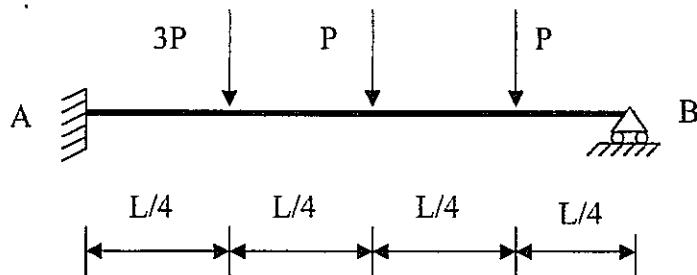


Table 1

Formulas for Beams

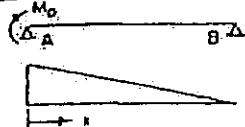
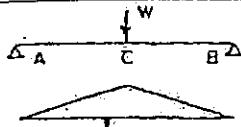
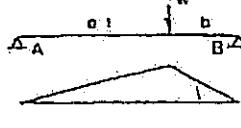
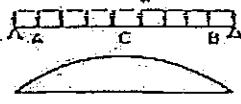
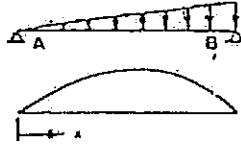
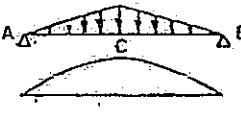
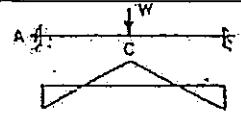
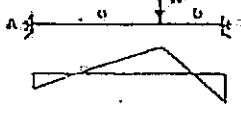
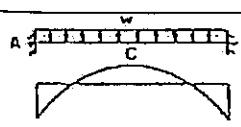
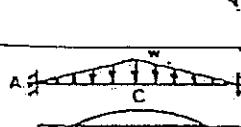
Structure	Shear 	Moment 	Slope 	Deflection 
Simply supported Beam				
	$S_A = -\frac{M_o}{L}$	M_o	$\theta_A = \frac{M_o L}{3EI}$ $\theta_B = -\frac{M_o L}{6EI}$	$Y_{max} = 0.062 \frac{M_o L^2}{EI}$ at $x = 0.422L$
	$S_A = \frac{W}{2}$	$M_o = \frac{WL}{4}$	$\theta_A = -\theta_B = \frac{WL^2}{16EI}$	$Y_c = \frac{WL^3}{48EI}$
	$S_A = \frac{Wb}{L}$ $S_B = \frac{Wa}{L}$	$M_o = \frac{Wab}{L}$	$\theta_A = \frac{Wab(L+b)}{6EI L}$ $\theta_B = -\frac{Wab(L+a)}{6EI L}$	$Y_o = \frac{Wa^2 b^2}{3EI L}$
	$S_A = \frac{WL}{2}$	$M_c = \frac{WL^2}{8}$	$\theta_A = -\theta_B = \frac{W L^3}{24EI}$	$Y_c = \frac{5WL^4}{384EI}$
	$S_A = \frac{WL}{6}$ $S_B = \frac{WL}{3}$	$M_{max} = 0.064 WL^2$ at $x = 0.577L$	$\theta_A = \frac{7WL^3}{360EI}$ $\theta_B = -\frac{8WL^3}{360EI}$	$Y_{max} = 0.00652 \frac{WL^4}{EI}$ at $x = 0.519L$
	$S_A = \frac{WL}{4}$	$M_c = \frac{WL^2}{12}$	$\theta_A = -\theta_B = \frac{5W L^3}{192EI}$	$Y_c = \frac{WL^4}{120EI}$
Fixed Beams				
	$S_A = \frac{W}{2}$	$M_c = \frac{WL}{8}$	$\theta_A = \theta_B = 0$	$Y_c = \frac{WL^3}{192EI}$
	$S_A = \frac{Wb^2}{L^3}(3a+b)$ $S_B = \frac{Wa^2}{L^3}(3b+a)$	$M_A = -\frac{Wab^2}{L^2}$ $M_B = -\frac{Wba^2}{L^2}$	$\theta_A = \theta_B = 0$	$Y_o = \frac{Wa^3 b^3}{3EI L^3}$
	$S_A = \frac{WL}{2}$	$M_A = M_B = -\frac{WL^2}{12}$	$\theta_A = \theta_B = 0$	$Y_c = \frac{WL^4}{384EI}$
	$S_A = \frac{3WL}{20}$ $S_B = -\frac{7WL}{20}$	$M_A = -\frac{WL^2}{30}$ $M_B = -\frac{WL^2}{20}$	$\theta_A = \theta_B = 0$	$Y_{max} = 0.00131 \frac{WL^4}{EI}$ at $x = 0.525L$
	$S_A = \frac{WL}{4}$	$M_A = M_B = -\frac{5WL^2}{96}$	$\theta_A = \theta_B = 0$	$Y_c = \frac{0.7WL^4}{384EI}$



Table 1

Formulas for Beams

Structure	Shear	Moment	Slope	Deflection
Cantilever Beam				
	0	M_o	$\theta_A = \frac{M_o L}{EI}$	$Y_A = \frac{M_o L^2}{2EI}$
	W	$M_B = -WL$	$\theta_A = -\frac{WL^2}{2EI}$	$Y_A = \frac{WL^3}{3EI}$
	$S_B = -WL$	$M_B = -\frac{WL^2}{2}$	$\theta_A = -\frac{WL^3}{6EI}$	$Y_A = \frac{WL^4}{8EI}$
	$S_B = -\frac{WL}{2}$	$M_B = -\frac{WL^2}{6}$	$\theta_A = -\frac{WL^3}{24EI}$	$Y_A = \frac{WL^4}{8EI}$
	$S_B = -\frac{WL}{2}$	$M_B = -\frac{WL^2}{2}$	$\theta_A = -\frac{WL^3}{8EI}$	$Y_A = \frac{11WL^4}{120EI}$
Propped Cantilever				
	$S_A = -\frac{3M_o}{2L}$	$M_B = -\frac{M_o}{2}$	$\theta_A = -\frac{M_o L}{4EI}$	$Y_{max} = \frac{M_o L^2}{27EI}$ at $x = \frac{L}{3}$
	$S_A = -\frac{3M_o}{2L}$	$M_B = -\frac{3WL}{16}$ $M_c = \frac{5WL}{32}$	$\theta_A = \frac{WL^2}{32EI}$	$Y_{max} = 0.00962 \frac{WL^3}{EI}$ at $x = 0.4471$
	$S_A = \frac{Wb^2}{2L^3} (a+2L)$ $S_B = -\frac{Wa}{2L^3} (3L^2 - a^2)$	$M_B = -\frac{Wab}{L^2} (a + \frac{b}{2})$	$\theta_A = \frac{Wab^2}{4EI L}$	$Y_0 = \frac{Wa^2 b^2}{12EI L^3} (3L +$
	$S_A = +\frac{3WL}{8}$	$M_B = -\frac{WL^2}{8}$	$\theta_A = \frac{WL^3}{48EI}$	$Y_{max} = 0.0054 \frac{WL^4}{EI}$ at $x = 0.421$
	$S_A = +\frac{WL}{10}$	$M_{max} = 0.03WL^2$ $at x = 0.447L$ $M_B = -\frac{WL^2}{15}$	$\theta_A = \frac{WL^3}{120EI}$	$Y_{max} = 0.00239 \frac{WL}{EI}$ at $x = 0.447$
	$S_A = +\frac{WL}{40}$	$M_{max} = 0.0423WL^2$ $at x = 0.329L$ $M_B = -\frac{7WL^2}{120}$	$\theta_A = \frac{WL^3}{80EI}$	$Y_{max} = 0.00305 \frac{WL}{EI}$ at $x = 0.402$