

# The Open University of Sri Lanka

## Faculty of Engineering Technology



Study Programme	: Bachelor of Technology Honours in Engineering
Name of the Examination	: Final Examination
Course Code and Title	: <b>DMX 6578 / MEX 6278 – FLUID MECHANICS</b>
Academic Year	: 2017/18
Date	: <b>16<sup>th</sup></b> , February 2019
Time	: 9.30am -12.30pm
Duration	: 3 hours

### General Instructions

1. Read all instruction carefully before answering the questions.
2. This question paper consists of 8 questions. All questions carry equal marks.
3. Answer **any 5** questions only.

### Q1.

- (a). By considering an infinitesimally small cubical fluid element, show that the general equation of continuity in three-dimensional space can be expressed as,

4 marks

$$\frac{\partial \rho}{\partial t} + \nabla \cdot (\rho \vec{V}) = 0$$

where ,

$$\nabla = \frac{\partial}{\partial x} + \frac{\partial}{\partial y} + \frac{\partial}{\partial z} \quad \text{and} \quad \vec{V} = u\vec{i} + v\vec{j} + w\vec{k}$$

- (b). By using the above relationship, obtain the continuity equation for the following fluid flows.

4 marks

- I. 2D - Compressible flow
- II. 3D- Steady - Incompressible flow

- (c). The velocity components for X and Y directions of a 3D-steady incompressible flow are  $u = x^2 + y^2z^3$  and  $v = -(xy + yz + zx)$  respectively. Find the missing component of the velocity distribution such that continuity equation is satisfied.

12 marks

Q2.

(a). By considering the forces acting on the differential control volume of a fluid, Write down the expression for the following terms in usual notations.

- I. Surface forces.
- II. Body forces.
- III. Material derivative.

2 marks

2 marks

2 marks

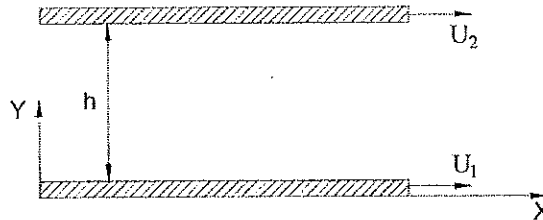


Figure Q2

(b). Show that the flow of a liquid per unit width ( $Q$ ) for laminar flow between two parallel plates moving in the same direction with different velocities, as shown in Figure Q2, is given by

10 marks

$$Q = \frac{h}{2}(U_1 + U_2) - \frac{1}{12\mu}h^3 \frac{dp}{dx}$$

(c). Derive an expression for the shear stress on top plate.

4 marks

Q3.

(a). Discuss the importance of Reynolds Number in relation to the behaviour of boundary layer.

2 marks

(b). Explain the significance of the boundary layer displacement thickness  $\delta^*$  and the momentum thickness  $\theta$ . Express  $\delta^*$  and  $\theta$  as integrals of the boundary layer velocity profiles on a smooth flat plate.

4 marks

(c). Calculate the ratio  $(\delta^* / \delta)$  for a laminar boundary layer with a velocity profile given by,

6 marks

$$\frac{u}{U_\infty} = 2\frac{y}{\delta} - \frac{y^2}{\delta^2}$$

(d). Show that the frictional drag force per unit width ( $F_d$ ) due to the boundary layer is given by,

8 marks

$$F_d = \frac{2\delta}{15} \rho U_\infty^2$$

Q4.

- (a). Briefly explain an application of the potential flow theory. 3 marks
- (b). State the conditions of a fluid flow that are required for the existence of a stream function and a velocity potential function. 3 marks
- (c). A fluid flow field is given by,

$$V = (6xt + yz^2)i + (3t + xy^2)j + (xy - 2xyz - 6tz)k$$

14 marks

Prove that the flow is steady incompressible fluid flow. Calculate the velocity and acceleration at the point (2,2,2) at  $t=2.0$ .

Q5.

- (a). By using suitable examples, explain dimensional homogeneity, geometric similarity, kinematic similarity and dynamic similarity. 3 marks
- (b). State Buckingham  $\pi$  theorem. 3 marks

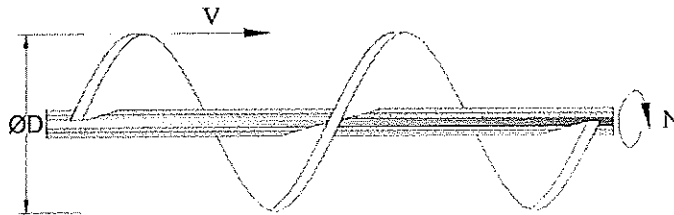


Figure Q5

- (c). The thrust (T) of a screw propeller (as shown in Figure Q5) is depends on the diameter (D), speed of advance (V), revolution per second (N), fluid density ( $\rho$ ) and the coefficient of viscosity ( $\mu$ ). Show that , 8 marks

$$T = \rho D^2 V^2 \phi \left( \frac{VD\rho}{\mu}, \frac{DN}{V} \right)$$

6 marks

- (d). The characteristics of a propeller of 4.8m diameter and rotational speed 120 rpm are examined by means of a geometrically similar model of 600mm diameter. When the model is run at 480 rpm by a torque of 30 Nm, the thrust developed is 300N and the speed of advance is  $3\text{ms}^{-1}$ . Determine the speed of advance (V) and Torque ( $\tau$ ) for the full scale propeller.

Efficiency ( $\eta$ ) of a propeller is given by 
$$\eta = \frac{\text{Output}}{\text{Input}} = \frac{TV}{\tau N}$$

Q6.

- (a). Write down the Bernoulli equation for 2D and state the assumptions that you have to make in order to apply it.

3 marks

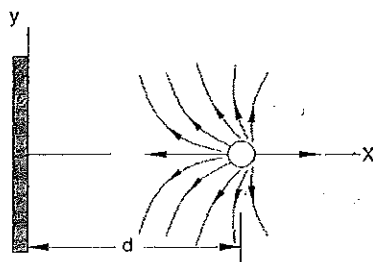


Figure Q6

- (b). A source of strength  $m$  is located at a distance  $d$  from a vertical solid wall as shown in Figure 3. The velocity potential for this incompressible, irrotational flow is given by

$$\phi = \frac{m}{4\pi} \{ \ln[(x-d)^2 + y^2] + \ln[(x+d)^2 + y^2] \}$$

- I. Show that there is no flow through the wall.
- II. Determine the velocity distribution along the wall.
- III. Determine the pressure distribution along the wall, assuming  $p=p_0$  far from the source.

5 marks

6 marks

6 marks

Q7.

- (a). Explain the importance of Laplace equation in relation to the potential flow theory.
- (b). What is the physical significance of a Rankine oval of equal axes?
- (c). A uniform flow of velocity  $U=10 \text{ ms}^{-1}$  in the positive x-direction is flowing over a doublet of strength  $\mu=15 \text{ m}^2\text{s}^{-1}$ . The doublet is in the line of the uniform flow. The polar coordinates of a point P in the flow field are  $0.9\text{m}$  and  $30^\circ$ . Determine,
  - I. Stream line function at point P
  - II. Resultant velocity at point P.

6 marks

6 marks

6 marks

8 marks

Stream function for a doublet in a uniform flow in usual notations is given by,

$$\psi = \left[ Ur - \frac{\mu}{2\pi r} \right] \sin \theta$$

Q8.

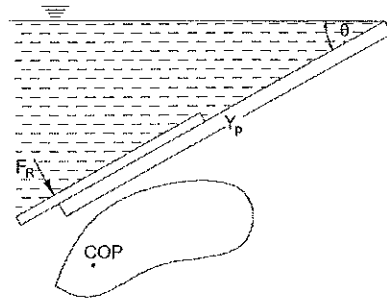


Figure Q8-a

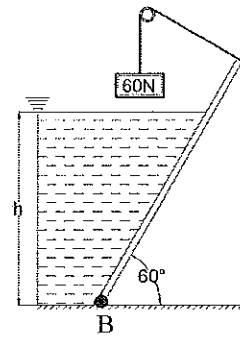


Figure Q8-b

- a. By considering the given Figure Q8-a, show that distance, from the free surface, to the center of pressure on an inclined immersed plane surface area is given by,

8 marks

$$y_p = \frac{\text{Second moment of area}}{\text{First moment of area}}$$

- b. A  $6\text{m} \times 2\text{m}$  rectangular gate hinged at the base (B) is inclined at an angle of  $60^\circ$  to the horizontal. The upper end of the gate is kept in position by a weight of  $60\text{kN}$  acting at angle of  $90^\circ$  as shown in Figure Q8-b. Neglecting the weight of the gate, compute the water level  $h$  for which the gate will start to fall.

12 marks

**Navier-Stokes equations for incompressible flow:-**

$$\rho \left( \frac{\partial u}{\partial t} + u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} + w \frac{\partial u}{\partial z} \right) = \rho g_x - \frac{\partial p}{\partial x} + \mu \left( \frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2} + \frac{\partial^2 u}{\partial z^2} \right)$$

$$\rho \left( \frac{\partial v}{\partial t} + u \frac{\partial v}{\partial x} + v \frac{\partial v}{\partial y} + w \frac{\partial v}{\partial z} \right) = \rho g_y - \frac{\partial p}{\partial y} + \mu \left( \frac{\partial^2 v}{\partial x^2} + \frac{\partial^2 v}{\partial y^2} + \frac{\partial^2 v}{\partial z^2} \right)$$

$$\rho \left( \frac{\partial w}{\partial t} + u \frac{\partial w}{\partial x} + v \frac{\partial w}{\partial y} + w \frac{\partial w}{\partial z} \right) = \rho g_z - \frac{\partial p}{\partial z} + \mu \left( \frac{\partial^2 w}{\partial x^2} + \frac{\partial^2 w}{\partial y^2} + \frac{\partial^2 w}{\partial z^2} \right)$$

**Frictional drag force per unit width due to laminar boundary layer :-**  $F_d = \rho \int_0^\delta u(U_\infty - u)dy$

**Acceleration vector:-**  $\vec{a} = a_x \mathbf{i} + a_y \mathbf{j} + a_z \mathbf{k}$  and  $\vec{a}_x = \frac{\partial u}{\partial t} + u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} + w \frac{\partial u}{\partial z}$

**Cartesian Coordinates:-**  $\nabla = \frac{\partial}{\partial x} + \frac{\partial}{\partial y} + \frac{\partial}{\partial z}$

**Cylindrical Coordinates:-**  $\nabla = \frac{\partial}{\partial r} + \frac{1}{r} \frac{\partial}{\partial \theta} + \frac{\partial}{\partial z}$   $V_r = \frac{1}{r} \frac{\partial \psi}{\partial \theta}$   $V_\theta = -\frac{\partial \psi}{\partial r}$

-----End-----

